Current and future energy calculation standards for heat pumps

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Laurent Socal Subtask C Leader, Annex 88 Consultant, Italy Friday 8th November 2024

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The context: building energy performance calculation methods

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A set of calculation procedures that take into account:

- The actual or specifically designed building envelope geometry and properties
- The actual or specifically designed HVAC, domestic hot water and lighting systems (comfort technical systems) properties
- A standardized set of climate and use conditions of the building (or actual data for audits)

... and **calculates** the overall energy performance of the building («weighted energy use», such as non-renewable primary energy, renewable primary energy, CO_2 emission, etc.) and other so called "partial performance indicators" (e.g. average transmittance, system efficiencies).

This is done by «modules», that is specific parts on each considered aspect

 \rightarrow the focus of this presentation is on the specific part that deals with the heat pump performance. The heat pump shall be calculated according to the specific operating conditions in the actual assessed building.

This includes also any «wrong» situation (e.g. oversized machines, bad building insulation).

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The big challenge: the sensitivity of heat pumps to operating conditions

Temperature of both sources

- Calculation of cold source temperature
 - External air temperature: hourly or bins for the monthly method
 - Surface/ground water temperature (heat exchanger approach?)
 - Ground heat exchanger outlet temperature (approach to ground? Ground temp drift?)
- Calculation of hot source temperature
 - Water based: calculation of required generator flow temperature (LWT)
 - Air based: indoor temperature or required duct air temperature (EAT)

Part load operation

- Continuous operation at reduced power (inverter) \rightarrow maximum turndown?
- ON-OFF operation on full load (ON-OFF control) or min load (inverter) \rightarrow cycling frequency
- Control options \rightarrow evaporator/condenser fan speed

Other influences

- Auxiliaries not included in the COP testing
 - Internal to the heat pump (absorption)
 - External (source pumps, etc.)
- Defrosting













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 Building rating Gives access to building market: building permit building on the market 	 Product rating Gives access to product market: product on the shelf
 Looks for the performance of each product in building specific and changing operating conditions 	 Looks for the performance of all products in the same operating conditions (representative ?)
 Needs parameters to assess the influence of each operating condition variable on the product performance 	 Needs a single performance indicator to rate the product on a scale
 Needs to distinguish between the effect of product characteristics and installation choices 	 Representative installation conditions may be embedded in the testing
 Has to foresee also the effect of bad products and wrong sizing, installation and operating conditions 	 No reference to possible wrong sizing, installation or operating conditions
EU EPBD Directive	EU Ecodesign + Ecolabeling Directives

Heat pump modules for building energy performance calculation The current challenge :

• Calculating the performance of the heat pump in a single calculation interval, which is a non-linear function of at least 3 variables: source and sink temperature, load

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- Additional factors: control strategy, impact of installation on cycling frequency, defrosting, transient operation during domestic hot water charge,
- Using the currently available data

 → several methods, approximations and default data to leverage the available data sets

Not satisfactory because of low accuracy and flattening of the evaluation of products within a building (you use the same defaults for all)

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Heat pump modules for building energy performance calculation

- Common approaches
 Starting from declared performance value in a single or several reference operating conditions, apply
- independent corrections for each influencing factor
 → basic assumption: the effect of each influencing factor is independent of the effect of the others
 Preprocess test results form several operating conditions to identify a characteristic function that describes
- the efficiency of the heat pump. Then apply the characteristic function in actual operating conditions. \rightarrow example: assume exergy efficiency is a function of required output capacity only
- The seasonal efficiency is assumed to be a weighted average of efficiency for a set of part loads.

Frequently used "additional" assumptions (for small adjustments and/or complementing)

- Constant exergy efficiency (heat pumps performance is a given fraction of Carnot theoretical maximum) This is used for interpolation and extrapolation when data for several testing points are available
- Approach of evaporator and condenser is a linear function of instantaneous capacity (to switch from available temperatures to relevant temperatures)

• .

Sample methods: EN 15316-4-2 (new draft, on-going review)

EN 15316-4-2 is structured as

- A main procedure, dealing with common issues like operating limits, priorities between services, required back-up, auxiliary energy, ...
- Specific parts according to two main calculation options ("path A" and "path B") options, designed to connect to two different sources of data.
 - Path A: intended to use data from EN 14511 \rightarrow capacity and COP at full load

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 Path B: intended to use data from EN 14825 → declared COP at various load and operating conditions

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 EN 15316-4-2 This is the backbone of the calculation procedure. The steps in orange Assessing the available capacity Calculating the COP in the calculation interval can be performed either with path A or path B depending on the heat pump type 	 Get product type (source / sink / technology) → decide calcul. Get product data Get required heat output for each service (heating and domess Calculate source temperature depending on source type Calculate sink temperature depending on service Get maximum available power output depending on operating Handle priority and load dispatch within the calculation intervice AA/B Correct performance (COP) according to actual operating constraints Calculate main energy input RA/B Repeat for all priorities Calculate back-up energy input and energy to be covered by not calculate auxiliary energy and recoverable losses 	ulation path (A /B and variant stic hot water) g conditions (per service) val ditions See next slides on Pa A and Path B detail next generator in the sequence	
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EN 15316-4-2

Path B basics

The source of the data is EN 14825.

Test points are selected assuming a perfect sizing o the heat pump and a strong flow temperature reset according to outdoor temperature or load.

Coverage of the performance range is limited.

Ongoing development: using data for several temperature levels and interpolating.

		Average climate					
Test point	T _{aria}	PLR	LWT «35»	LWT «55»			
	°C		°C	°C			
Α	-7	0,88	34	52			
В	2	0,54	30	42			
С	7	0,35	27	36			
D	12	0,15	24	30			

Definition of testing points according to EN 14825 for average climate and 2 temperature levels.

Part load test are static tests. Test load is $P_{design} \times PLR$



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Comparison between testing points according to EN 14825 for average climate and 35 °C and a full performance map of a heat pump

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UNI-TS 11300

Italy

Same method as path A of EN 15316-4-2.

Correction according to part load is based on a simple default curve.

Required data point are listed in a table that includes summer air temperature because of domestic hot water production.

Not satisfactory with air to air het pumps and domestic hot water heaters.

T	Reference source			Reference sink temperatures, $\theta_{snk;ref,i}$						
Type of source	temperatures θ _{src;ref,j}		Air	Water		Domestic hot water				
External air	-7	2	7	12						
Surface water and ground water		5	10	15	20	35	45	55	45	55
Ground heat exchanger	-5	0	5	10						
Domestic hot water heaters, air source only	7	15	20	35	Not a	pplicat	ole		45	55



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RE 2020 France Same method as path A of EN 15316-4-2. The full load performance map is generated from one single value according to default multipliers to fill in the grid. COP is interpolated according to actual source and sink temperature then the correction according to part load is applied	COP _{LR100} 25 35 45 55 65 f _{COP;src}	-15 ▲ 1.94 1.76 ← ♥ 1.41 ♥ 1.13 ♥ 0.90 0.80 A-7 to A-15 A set of c each type	-7 2,42 2,20 ← 1,76 1,41 1,13 0,625 A2 to A-7 default mul e of heat p	θsrc 2 3,87 3,52 ← 2,82 2,25 1,80 0,80 A7 to A2 tipliers is de ump (AA / /	7	20 6,05 → 5,50 4,40 3,52 2,82 1,25 A7 to A20	1,10 0,80 0,80 0,80	fcoP:snk W35 to W25 W35 to W45 W45 to W55 W55 to W65 Default multipliers
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SAP 10.2 United Kingdom

For each product, manufacturer shall perform an hourly calculation to determine a seasonal SCOP that will be uploaded to an official data-base for use by assessors.

The seasonal SCOP is calculated for a number of heat emitters and for a range of relative sizing of the heat pump compared to the building needs (PSR = Plant sizing ratio)

PSR (Plant Size Ratio)	0,2	0,5	0,8	1,0	1,2	1,5	2,0
Floor heating							
Heating SPF	3,855	3,926	4,173	4,217	4,241	4,191	4,108
Running hours	4925	2716	1770	1447	1231	1016	800
Radiators							
Heating SPF	3.502	3.413	3.607	3.665	3.698	3.657	3.585
Running hours	4699	2516	1643	1345	1146	947	749
Convectors							
Heating SPF	3.930	3.89	4.133	4.186	4.215	4.166	4.081
Running hours	4776	2577	1682	1376	1172	968	764

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... or you have to perform curve fitting ...

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NECB (Canada) The method is similar to Energy plus An example of default performance curve for air source heat pump is shown bere	$Q_{available} = Q_{rated} \times CAP_FT_{EAS}$ $CAP_FT_{EAS} = a + (b \times t_{odb}) + (c \times t_{odb}^2) + a = 0.2536714 \qquad c = -0.0001862 \\ b = 0.0104351 \qquad d = 0.0000015$ $P_{operating} = P_{rated} \times EIR_FPLR \times EIR_FT$	$(d \times t_{odb}^{3})$ $(d \times t_{odb}^{3})$ $(d \times cap_FT_{EAS})$	t _{odb} = outdoor dry bulb temperature in °F
NOTE: there is no correction based on the leaving water temperature level. A correlation with outdoor dry bulb temperature is assumed.	$EIR_FPLR = a + (b \times PLR) + (c \times PLR^{2}) + (a = 0.0856522 c = -0.1834366 b = 0.9388137 d = 0.1589702 EIR_FT = a + (b \times t_{odb}) + (c \times t_{odb}^{2}) + (d = 0.0008800) b = -0.0622539 d = -0.0000044 d = -0.00000044 d = -0.0000044 d = -0.0000040 d = -0.0000044 d = -0.000004 d = -0.000004 $	$+ (d \times PLR^{3})$ 1 $d \times t_{odb}^{3}$ 6	PLR = part load ratio based on available capacity t _{odb} = outdoor dry bulb temperature in °F

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Conclusion

- Heat pumps are damn sensitive to operating conditions: 3 and non linear with a result in the range 1 to 3
- There is a huge variety of heat pumps typologies depending on source type, sink type, control options, combination with other products and services, etc.

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- The product standardization has focused mainly on the product rating
- Countries have different prevailing needs (heating versus cooling) and different type of prevailing systems (air based versus water based)
- Building energy performance calculation requires the coverage of very various situations, often far away from the assumptions used e.g. in product rating.
- Given the variety of heat pump types and in the absence of well defined product data intended for energy performance calculation, all sorts of calculation methods have developed, none comprehensive and fully satisfactory
- Input data and methods are interdependent: much more collaboration is needed between heat pump experts and building energy performance experts to find a good compromise between the required effort (testing several operating conditions to map the performance of the heat pump) and the benefit (exhaustive and accurate prediction of the heat pump performance in all foreseeable operating conditions)
- New typologies are emerging, VRV/VRF, desuperheating, simultaneous heating and cooling, integration with ventilation...
- Huge work in progress...